



Jet Aircraft Propulsion

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Lect-14

In this lecture...

- Axial flow compressors
 - Basic operation of axial compressors
 - Velocity triangles
 - Work and compression
 - Design parameters
 - Flow coefficient
 - Loading coefficient
 - Degree of reaction
 - Diffusion factor

Basic operation of axial compressors

- Axial flow compressors usually consists of a series of stages.
- Each stage comprises of a row of rotor blades followed by a row of stator blades.
- The working fluid is initially accelerated by the rotor blades and then decelerated in the stator passages.
- In the stator, the kinetic energy transferred in the rotor is converted to static pressure.
- This process is repeated in several stages to yield the necessary overall pressure ratio.

Basic operation of axial compressors

- The compression process consists of a series of diffusions.
- This occurs both in the rotor as well as the stator.
- Due to motion of the rotor blades → two distinct velocity components: absolute and relative velocities in the rotor.
- The absolute velocity of the fluid is increased in the rotor, whereas the relative velocity is decreased, leading to diffusion.
- Per stage pressure ratio is limited because a compressor operates in an adverse pressure gradient environment.

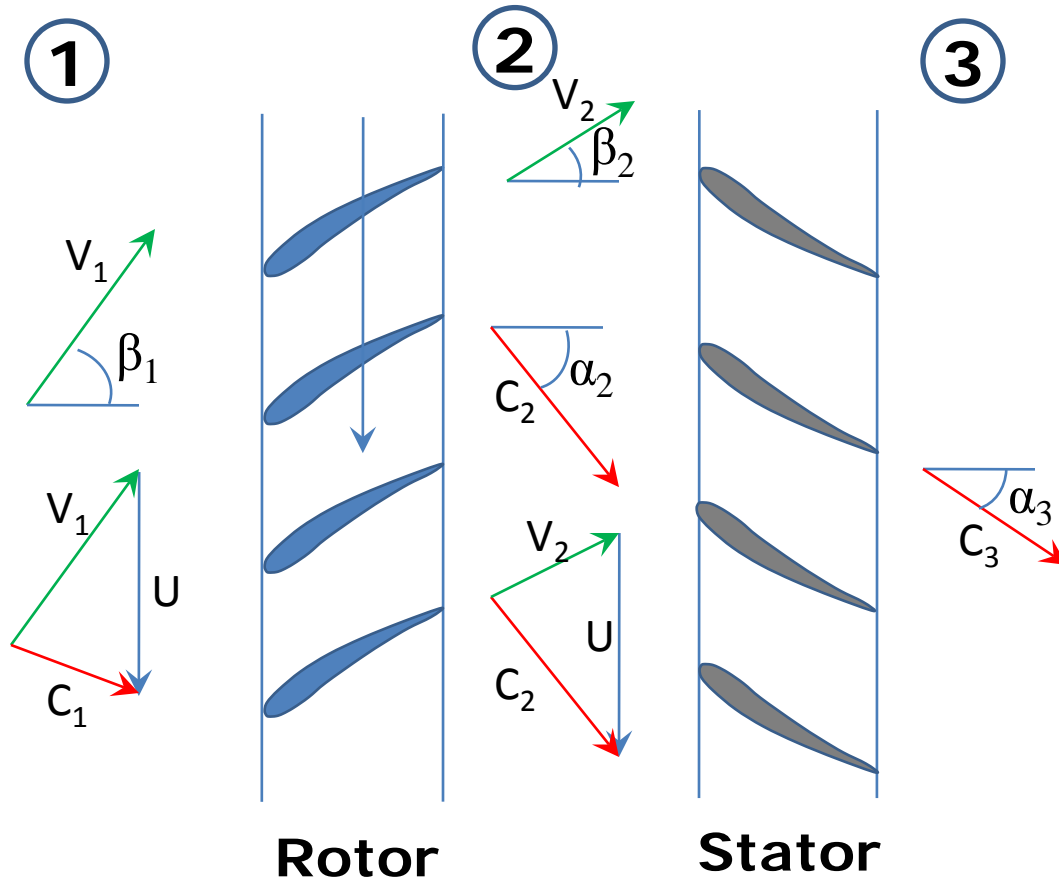
Basic operation of axial compressors

- Turbines on the other hand operate under favourable pressure gradients.
- Several stages of an axial compressor can be driven by a single turbine stage.
- Careful design of the compressor blading is essential to minimize losses as well as to ensure stable operation.
- Some compressors also have inlet Guide Vanes (IGV) that permit the flow entering the first stage to vary under off-design conditions.

Velocity triangles

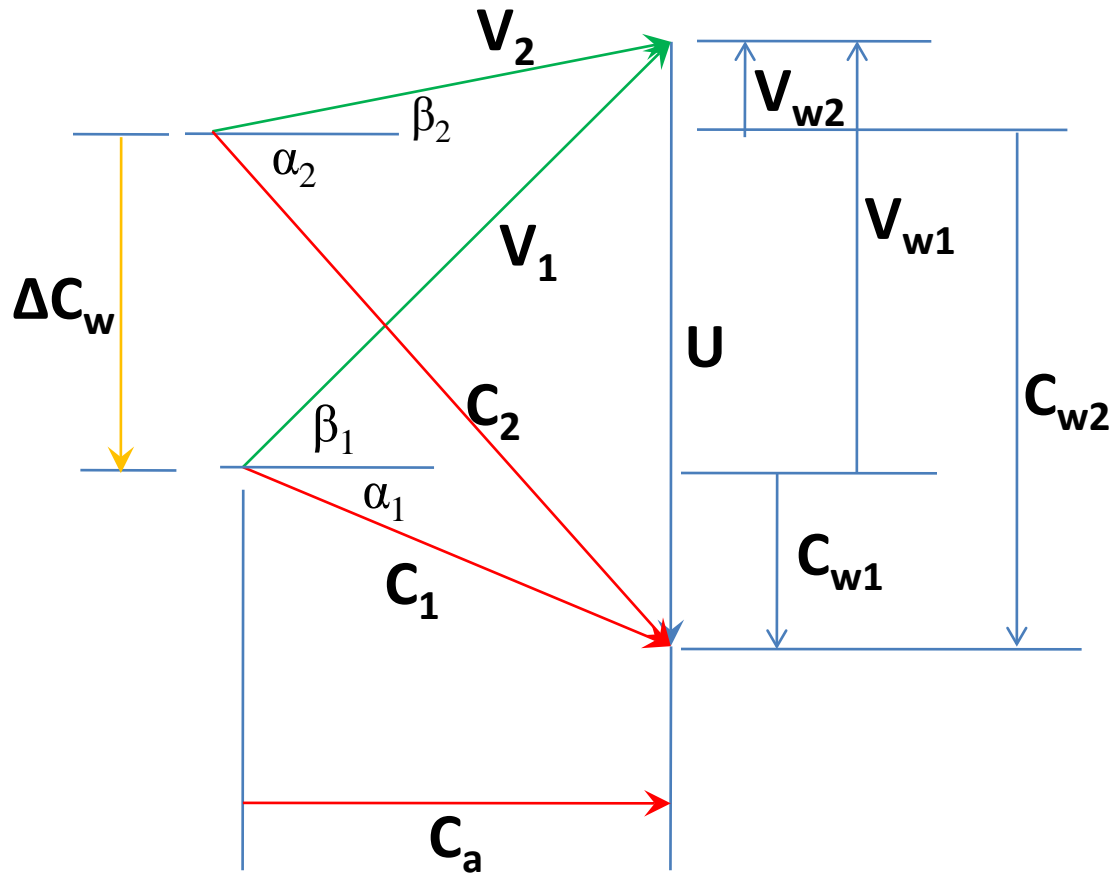
- Elementary analysis of axial compressors begins with velocity triangles.
- The analysis will be carried out at the mean height of the blade, where the peripheral velocity or the blade speed is, U .
- The absolute component of velocity will be denoted by, C and the relative component by, V .
- The axial velocity (absolute) will be denoted by C_a and the tangential components will be denoted by subscript w (for eg, C_w or V_w)
- α denotes the angle between the absolute velocity with the axial direction and β the corresponding angle for the relative velocity.

Velocity triangles

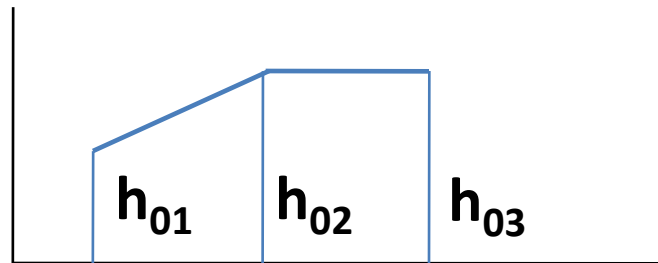


$$\vec{C} = \vec{U} + \vec{V}$$

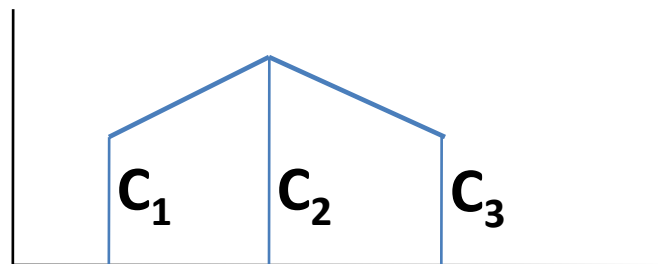
Velocity triangles



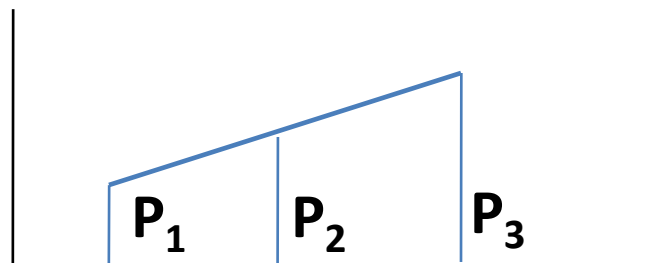
Property changes across a stage



Total enthalpy



Absolute velocity



Static pressure

Rotor Stator

Work and compression

- Assuming $C_a = C_{a1} = C_{a2}$, from the velocity triangles, we can see that

$$\frac{U}{C_a} = \tan \alpha_1 + \tan \beta_1 \quad \text{and} \quad \frac{U}{C_a} = \tan \alpha_2 + \tan \beta_2$$

- By considering the change in angular momentum of the air passing through the rotor, work done per unit mass flow is

$w = U(C_{w2} - C_{w1})$, where C_{w1} and C_{w2} are the tangential components of the fluid velocity before and after the rotor, respectively.

Work and compression

The above equation can also be written as,

$$w = UC_a (\tan \alpha_2 - \tan \alpha_1)$$

Since, $(\tan \alpha_2 - \tan \alpha_1) = (\tan \beta_1 - \tan \beta_2)$

$$\therefore w = UC_a (\tan \beta_1 - \tan \beta_2)$$

In other words, $w = U\Delta C_w$

- The input energy will reveal itself in the form of rise in stagnation temperature of the air.
- The work done as given above will also be equal to the change in stagnation enthalpy across the stage.

Work and compression

$$h_{02} - h_{01} = U\Delta C_w$$

$$T_{02} - T_{01} = \frac{U\Delta C_w}{c_p} \Rightarrow \frac{\Delta T_0}{T_{01}} = \frac{U\Delta C_w}{c_p T_{01}}$$

Since the flow is adiabatic and no work is done as the fluid passes through the stator, $T_{03} = T_{02}$

Let us define stage efficiency, η_{st} , as

$$\eta_{st} = \frac{h_{03s} - h_{01}}{h_{03} - h_{01}}$$

This can be expressed as

$$\frac{T_{03s}}{T_{01}} = 1 + \eta_{st} \frac{\Delta T_0}{T_{01}}$$

Work and compression

In the above equation, $\Delta T_0 = T_{03} - T_{01}$

In terms of pressure ratio,

$$\frac{P_{03}}{P_{01}} = \left[1 + \eta_{st} \frac{\Delta T_0}{T_{01}} \right]^{\gamma/(\gamma-1)}$$

This can be combined with the earlier equation to give,

$$\frac{P_{03}}{P_{01}} = \left[1 + \eta_{st} \frac{U \Delta C_w}{c_p T_{01}} \right]^{\gamma/(\gamma-1)}$$

Work and compression

- From the above equation that relates the per stage temperature rise to the pressure ratio, it can be seen that to obtain a high temperature ratio for a given overall pressure ratio (for minimizing number of stages),
 - High blade speed: limited by blades stresses
 - High axial velocity, high fluid deflection
(β_1 - β_2): Aerodynamic considerations and adverse pressure gradients limit the above.

Design parameters

- The following design parameters are often used in the parametric study of axial compressors:
 - Flow coefficient,
$$\phi = C_a / U$$
 - Stage loading,
$$\psi = \Delta h_0 / U^2 = \Delta C_w / U$$
 - Degree of reaction, R_x
 - Diffusion factor, D^*

Degree of reaction

- Diffusion takes place in both rotor and the stator.
- Static pressure rises in the rotor as well as the stator.
- Degree of reaction provides a measure of the extent to which the rotor contributes to the overall pressure rise in the stage.

Degree of reaction

$$R_x = \frac{\text{Static enthalpy rise in the rotor}}{\text{Stagnation enthalpy rise in the stage}}$$

$$= \frac{h_2 - h_1}{h_{03} - h_{01}} \approx \frac{h_2 - h_1}{h_{02} - h_{01}}$$

For a nearly incompressible flow,

$$h_2 - h_1 \cong \frac{1}{\rho} (P_2 - P_1) \text{ for the rotor}$$

$$\text{and for the stage, } h_{03} - h_{01} \cong \frac{1}{\rho} (P_{03} - P_{01})$$

$$\therefore R_x = \frac{h_2 - h_1}{h_{02} - h_{01}} \cong \frac{P_2 - P_1}{P_{02} - P_{01}}$$

Degree of reaction

From the steady flow energy equation,

$$h_1 + \frac{V_1^2}{2} = h_2 + \frac{V_2^2}{2}$$

$$\therefore R_x = \frac{h_2 - h_1}{h_{03} - h_{01}} = \frac{V_1^2 - V_2^2}{2U(C_{w2} - C_{w1})}$$

For constant axial velocity, $V_1^2 - V_2^2 = V_{w1}^2 - V_{w2}^2$

And, $V_{w1} - V_{w2} = C_{w1} - C_{w2}$

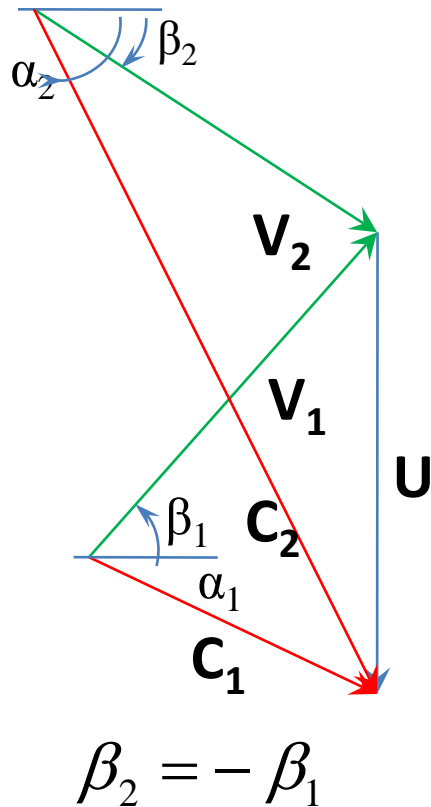
On simplification, $R_x = \frac{1}{2} - \frac{C_a}{2U} (\tan \alpha_1 - \tan \beta_2)$

or, $R_x = \frac{C_a}{2U} (\tan \beta_1 + \tan \beta_2)$

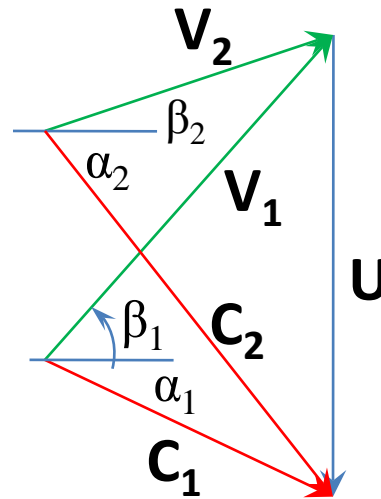
Degree of reaction

- Special cases of R_x
 - $R_x=0, \beta_2 = -\beta_1$, There is no pressure rise in the rotor, the entire pressure rise is due to the stator, the rotor merely deflects the incoming flow: impulse blading
 - $R_x=0.5$, gives $\alpha_1 = \beta_2$ and $\alpha_2 = \beta_1$, the velocity triangles are symmetric, equal pressure rise in the rotor and the stator
 - $R_x=1.0, \alpha_2 = -\alpha_1$, entire pressure rise takes place in the rotor while the stator has no contribution.

Degree of reaction

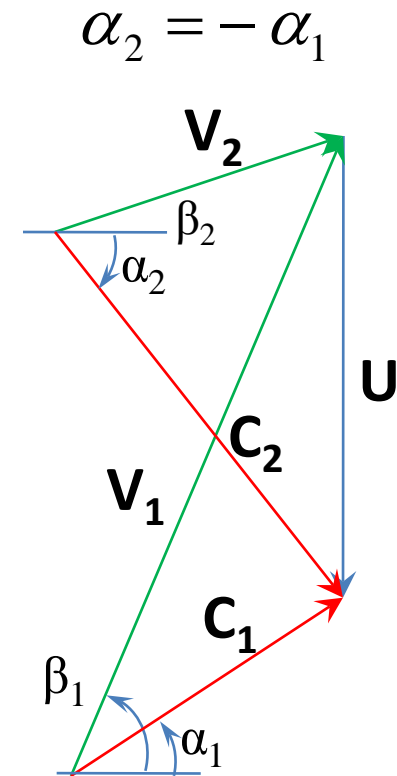


$R_x = 0.0$



$\alpha_1 = \beta_2$ and $\alpha_2 = \beta_1$

$R_x = 0.5$



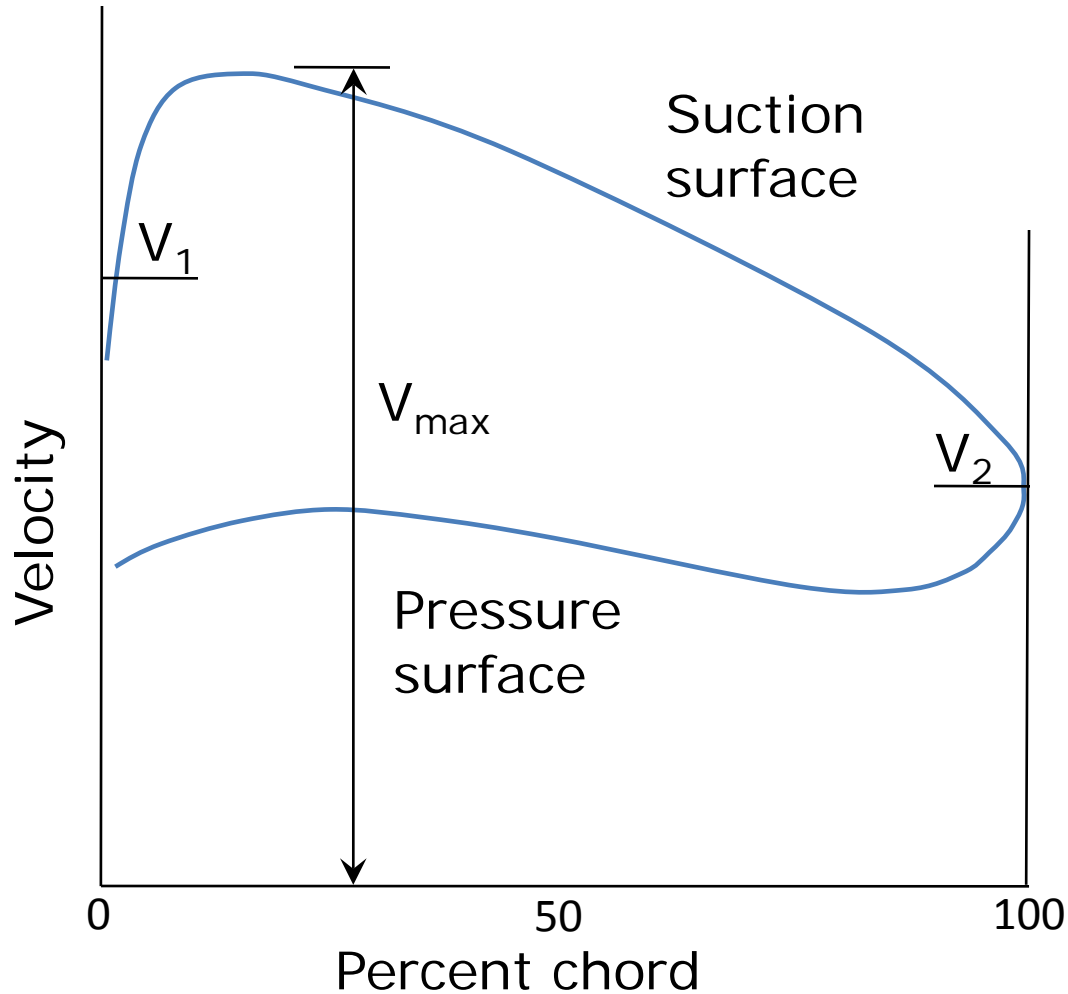
$R_x = 1.0$

Diffusion factor

- Fluid deflection ($\beta_2 - \beta_1$) is an important parameter that affects the stage pressure rise.
- Excessive deflection, which means high rate of diffusion, will lead to blade stall.
- Diffusion factor is a parameter that associates blade stall with deceleration on the suction surface of the airfoil section.
- Diffusion factor, D^* , is defined as

$$D^* = \frac{V_{\max} - V_2}{V_1} \quad \text{Where, } V_{\max} \text{ is the ideal surface velocity at the minimum pressure point and } V_2 \text{ is the ideal velocity at the trailing edge and } V_1 \text{ is the velocity at the leading edge.}$$

Diffusion factor



Diffusion factor

- Lieblein (1953) proposed an empirical parameter for diffusion factor.
 - It is expressed entirely in terms of known or measured quantities.
 - It depends strongly upon solidity (C/s).
 - It has been proven to be a dependable indicator of approach to separation for a variety of blade shapes.
 - D^* is usually kept around 0.5.

$$D^* = 1 - \frac{V_2}{V_1} + \frac{V_{w1} - V_{w2}}{2\left(\frac{C}{s}\right)V_1}$$

Where, C is the chord of the blade and s is the spacing between the blades.

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In the next lecture...

- Cascade analysis
 - Cascade nomenclature
 - Loss and blade performance estimation