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TURBOMACHINERY AERODYNAMICS

In this lecture...

- Turbine Blade Cooling
 - Blade cooling requirements
 - Fundamentals of heat transfer

- For a given pressure ratio and adiabatic efficiency, the turbine work per unit mass is proportional to the inlet stagnation temperature.
- Therefore, typically a 1% increase in the turbine inlet temperature can cause 2-3% increase in the engine output.
- Therefore there are elaborate methods used for cooling the turbine nozzle and rotor blades.
- Turbine blades with cooling can withstand temperatures higher than that permissible by the blade materials.

- Thrust of a jet engine is a direct function of the turbine inlet temperature.
- Brayton cycle analysis, effect of maximum cycle temperature on work output and efficiency.
- Materials that are presently available cannot withstand a temperature in excess of 1300 K.
- However, the turbine inlet temperature can be raised to temperatures higher than this by employing blade cooling techniques.
- Associated with the gain in performance is the mechanical, aerodynamic and thermodynamic complexities involved in design and analysis of these cooling techniques.

- The environment in which the nozzles and rotors operate are very extreme.
- In addition to high temperatures, turbine stages are also subjected to significant variations in temperature.
- The flow is unsteady and highly turbulent resulting in random fluctuations in temperatures.
- The nozzle is subjected to the most severe operating conditions.

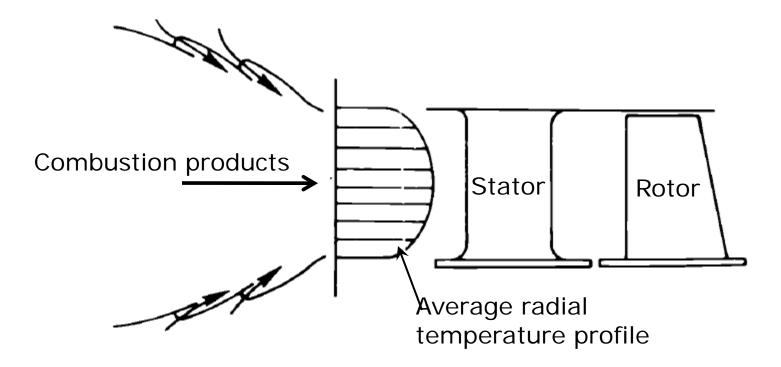
- Because the relative Mach number that the rotor experiences, it perceives lower stagnation temperatures (about 200-300 K) than the nozzle.
- However the rotor experience far more stresses due to the high rotational speeds.
- The highest temperatures are felt primarily by the first stage.
- Cooling problems are less complicated in later stages of the turbine.

- There are several modes of failure of a turbine blade.
 - Oxidation/erosion/corrosion
 - Occurs due to chemical and particulate attack from the hot gases.
 - Creep
 - Occurs as a result of prolonged exposure to high temperatures.
 - Thermal fatigue
 - As a result of repeated cycling through high thermal stresses.



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Turbine blade cooling



Average temperature profile entering a turbine stage

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- Turbine blade cooling involves application of concepts of heat transfer.
- Heat transfer is a well established area and substantial knowledge base is available in the form of books, journals and other forms of literature.
- We shall take a brief overview of the concepts of heat transfer that are required for understanding of the problems involved in turbine blade cooling.

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- There are three modes of heat transfer
 - Conduction
 - Convection
 - Radiation
- Conduction
 - Heat transfer between two bodies or two parts of the same body through molecules which are more or less stationary.
 - In liquids and gases, conduction results from transport of energy by molecular motion near the walls and in solids it takes place by a combination of lattice vibration and electron transport.

- Conduction involves energy transfer at a molecular level with no movement of macroscopic portions of matter relative to one another.
- Convection
 - Involves mass movement of fluids
 - When temperature difference produces a density difference – leads to mass movement – Free convection
 - Caused by external devices like a pump, blower etc. Forced convection

- Radiation
 - Energy transfer taking place through electromagnetic waves
 - Radiation does not require a medium
- For the temperatures that are encountered in a turbine, conduction and convection are the major modes of heat transfer.
- Radiative heat transfer is usually negligible and is normally not considered in turbine heat transfer analysis.

Fundamentals of heat transfer

- Heat transfer by conduction
 - The rate of heat transfer by conduction can be written as (Fourier's conduction law)

$$\frac{Q}{A} = q = -k \frac{dT}{dy}$$

Where, Q / A is the rate of heat transfer per unit area of the surface, and dT/dy is the temperature gradient. k is the thermal conductivity defined as the amount of heat conducted per unit time per unit area per unit negative temperature gradient.

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Fundamentals of heat transfer

The generalized governing equation is a three dimensional Poisson equation

$$\frac{\mathsf{k}}{\rho\mathsf{c}_{\mathsf{p}}}\nabla^{2}\mathsf{T} = \frac{\partial\mathsf{T}}{\partial\mathsf{t}}$$

This is known as the Fourier equation. The parameter $\frac{\kappa}{\rho c_p}$ is called thermal diffusivity and is a property of the conducting material.

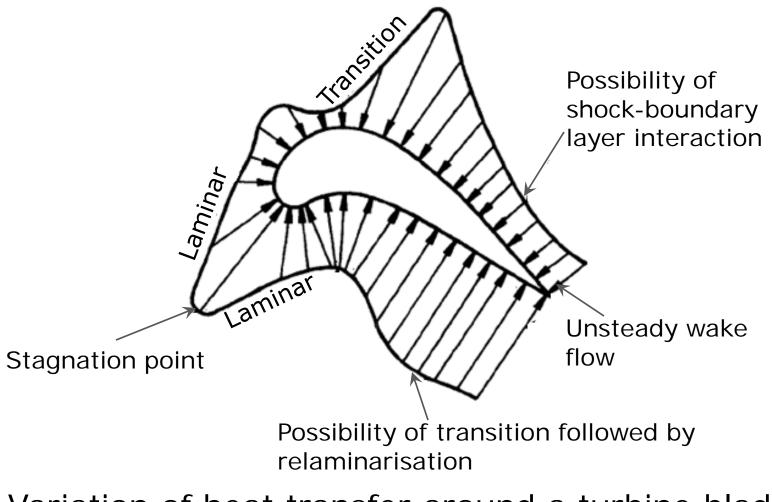
Simplified forms of this equation has been used extensively over the years by several researchers.

- Heat transfer by convection
 - Unlike in a solid, heat transfer in a fluid can take place through conduction as well as convection.
 - In general, the temperature and velocity fields are coupled and have strong influence on each other.
 - In modern day turbines, velocity as well as temperature gradients are high.
 - Forced convection is the dominant phenomena in turbine flows.

- In a typical turbine blade, the boundary layer developing on the blade surface and the freestream temperature are of interest.
- The boundary layer that acts as a buffer between the solid blade and the hot freestream, offers resistance to heat transfer.
- Heat transfer occurs in this viscous layer between the blade and the fluid through both conduction and convection.
- The nature of the boundary layer (laminar or turbulent) plays an important role in the heat transfer process.

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Fundamentals of heat transfer



Variation of heat transfer around a turbine blade

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Fundamentals of heat transfer

- Due to close coupling between fluid mechanics and heat transfer, each of the regions around a blade require special analysis valid for that region.
- The overall heat transfer is related to the temperature difference between the fluid and the solid through the Newton's law of cooling:

$$q_w(x) = h(x)(T_r - T_w) = k \left(\frac{\partial T}{\partial y}\right)_w$$

where, $q_w(x)$ is the heat flux from the fluid to the wall, h(x) is the heat transfer coefficient.

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Fundamentals of heat transfer

 The heat transfer coefficient is nondimensionalised by the thermal conductivity and characteristic length:

$$Nu_x = \frac{h(x)L}{k} = \frac{L}{T_e - T_w} \left(\frac{\partial T}{\partial y}\right)_w$$

 Nu_x is the Nusselt number.

- In addition to Nusselt number there are other important non-dimensional groups namely, Reynolds number (Re), Prandtl number (PR), Eckert's number (Ec), Grashof number (Gr) Richardson number (Ri) and Stanton number (St).
- All these numbers play a significant role in a transfer analysis depending upon the application.

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Laminar boundary layer (forced convection)

Consider an incompressible laminar flow over a flat plate. We can write the transport equation for such a case as :

$$\frac{\partial(\mathbf{u}\phi)}{\partial \mathbf{x}} + \frac{\partial(\mathbf{v}\phi)}{\partial \mathbf{y}} = \alpha \frac{\partial^2 \phi}{\partial \mathbf{y}^2}$$

$$\omega = \mathbf{u} \text{ or } \theta \quad \alpha = \mathbf{u} / \alpha \text{ or } \mathbf{k} / \alpha = \alpha$$

where, $\varphi = u \text{ or } \theta$, $\alpha = \mu / \rho \text{ or } k / \rho c_p$ and $\theta = (T - T_w) / (T_e - T_w)$

The boundary conditions being :

y = 0, $\phi = v = 0$ and $y \rightarrow \infty$, $\phi = u = \theta = 1$

 The transport equations for velocity and temperature are similar and therefore the coupling is obvious.

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Laminar boundary layer (forced convection)

 It can be shown that the heat transfer is related to the Reynolds number and Prandtl number through the Nusselt number.

 $Nu_x = 0.332(Re_x)^{1/2}(PR)^{1/3} = \frac{C_f}{2}(PR)^{1/3}Re_x$

- Heat transfer is a function of $(Re_x)^{1/2}$ and $PR^{1/3}$ and C_f .
- A thin boundary layer has a larger heat transfer.
- Therefore maximum heat transfer in a turbine blade occurs near the stagnation point and the leading edge.

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Turbulent boundary layer (forced convection)

• The heat transfer due to turbulent fluctuations is written as:

 $q_t = \rho c_p \overline{v' \, T'} = -c_p \epsilon_t \, \frac{\partial T}{\partial y} \ \text{ where, } \epsilon_t \text{ is the eddy diffusivity.}$

- There is a close coupling between the momentum transfer and heat transfer, which in turn translates to coupling between heat flux and shear stress.
- We can therefore define the turbulent Prandtl number as

$$PR_t = \frac{\mu_t}{\varepsilon_t}$$

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Turbulent boundary layer (forced convection)

Hence the ratio of heat flux and momentum flux is given by

$$\frac{q_{t}}{\tau_{t}} = -\frac{c_{p}(\partial T / \partial y)}{PR_{t}(\partial u / \partial y)}$$

The total rate of heat transfer due to both molecular and turbulent motions is

$$q = q_{molecular} + q_{turbulent} = -c_{p} \left(\frac{\mu}{PR} + \frac{\mu_{t}}{PR_{t}} \right) \frac{\partial T}{\partial y}$$

There is a clear difference between PR and PR_t . The Prandtl number (PR) is a physical property of the fluid, whereas the Turbulent Prandlt number (PR_t) is a property of the flowfield.

TURBOMACHINERY AERODYNAMICSLect-27Turbulent boundary layer (forced
convection)

For a flat plate with a turbulent boundary layer, the following equation is commonly used :

$$Nu_x = 0.029(Re_x)^{4/5}PR^{1/3}$$

A general equation for both laminar and turbulent flow analysis can be written as $Nu_x = A \operatorname{Re}_x^m \operatorname{PR}^n$ where, A, m and n are constants for a particular flow. This is called the Nusselt's equation.

- Based on our discussion on laminar and turbulent flows:
 - Heat transfer is higher for a thin boundary layer than a thick boundary layer as the temperature gradient is higher for a thin boundary layer.
 - Heat transfer for a turbulent boundary layer is higher than a laminar boundary layer.
 - Heat transfer in thin viscous regions like stagnation point or leading edge, is very high. The velocity and temperature gradients are extremely high in these zones.

- In order to decide the cooling methodology to be used in a turbine blade, a very strong understanding of the heat transfer mechanisms are essential.
- Turbine blade cooling requires significant amount of compressor air (as high as 20%).
- The cooling air also mixes with the turbine flow leading to losses.
- Due to the above, vigorous analysis is carried out to minimize the amount of cooling as well as the negative aerodynamic effects of cooling.

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