

Lecture – 21

# *Pole Placement Control Design*

*Dr. Radhakant Padhi*

*Asst. Professor*

*Dept. of Aerospace Engineering*

*Indian Institute of Science - Bangalore*



# Pole Placement Control Design

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## **Assumptions:**

- The system is completely state controllable.
- The state variables are measurable and are available for feedback.
- Control input is unconstrained.

# Pole Placement Control Design

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## **Objective:**

The closed loop poles should lie at  $\mu_1, \dots, \mu_n$ , which are their ‘desired locations’.

## **Difference from classical approach:**

Not only the “dominant poles”, but “all poles” are forced to lie at specific desired locations.

## **Necessary and sufficient condition:**

The system is completely state controllable.

## Closed Loop System Dynamics

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$$\dot{X} = AX + BU$$

The control vector  $U$  is designed in the following state feedback form

$$U = -KX$$

This leads to the following closed loop system

$$\dot{X} = (A - BK)X = A_{CL}X$$

where  $A_{CL} \triangleq (A - BK)$

# Philosophy of Pole Placement Control Design

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The gain matrix  $K$  is designed in such a way that

$$\left|sI - (A - BK)\right| = (s - \mu_1)(s - \mu_2) \cdots (s - \mu_n)$$

where  $\mu_1, \dots, \mu_n$  are the desired pole locations.

## Pole Placement Design Steps: Method 1 (low order systems, $n \leq 3$ )

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- Check controllability
- Define  $K = [k_1 \quad k_2 \quad k_3]$
- Substitute this gain in the desired characteristic polynomial equation

$$|sI - A + BK| = (s - \mu_1) \cdots (s - \mu_n)$$

- Solve for  $k_1, k_2, k_3$  by equating the like powers on both sides

## Pole Placement Control Design: Method - 2

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$$\dot{X} = AX + Bu$$

$$u = -KX, \quad K = [k_1 \ k_2 \ \cdots \ k_n]$$

**Let the system be in first companion (controllable canonical) form**

$$A = \begin{bmatrix} 0 & 1 & 0 & 0 & \cdots & 0 \\ 0 & 0 & 1 & 0 & \cdots & 0 \\ \vdots & & & \ddots & & \vdots \\ 0 & 0 & 0 & 0 & \ddots & 1 \\ -a_n & -a_{n-1} & -a_{n-2} & -a_{n-3} & \cdots & -a_1 \end{bmatrix}, B = \begin{bmatrix} 0 \\ 0 \\ \vdots \\ 0 \\ 1 \end{bmatrix}$$

After applying the control, the closed loop system dynamics is given by

$$\dot{X} = (A - BK)X = A_{CL}X$$

$$A_{CL} = \begin{bmatrix} 0 & 1 & 0 & \cdots & 0 \\ 0 & 0 & 1 & \cdots & 0 \\ \vdots & & \ddots & & \\ \vdots & & & \ddots & 1 \\ 0 & 0 & 0 & \cdots & 0 \\ -a_n & -a_{n-1} & -a_{n-2} & \cdots & -a_1 \end{bmatrix} - \begin{bmatrix} 0 & 0 & 0 & \cdots & 0 \\ 0 & 0 & 0 & \cdots & 0 \\ \vdots & & \ddots & & \vdots \\ \vdots & & & \ddots & \vdots \\ 0 & 0 & 0 & \cdots & 0 \\ k_1 & k_2 & k_3 & \cdots & k_n \end{bmatrix}$$

$$A_{CL} = \begin{bmatrix} 0 & 1 & 0 & \cdots & 0 \\ 0 & 0 & 1 & \cdots & 0 \\ 0 & 0 & 0 & \cdots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & 0 & 0 & \cdots & 1 \\ (-a_n - k_1) & (-a_{n-1} - k_2) & \cdots & \cdots & (-a_1 - k_n) \end{bmatrix} \dots\dots\dots (1)$$



## Pole Placement Control Design: Method - 2

If  $\mu_1, \dots, \mu_n$  are the desired poles. Then the desired characteristic polynomial is given by,

$$(s - \mu_1) \cdots (s - \mu_n) = s^n + \alpha_1 s^{n-1} + \alpha_2 s^{n-2} + \cdots + \alpha_n$$

This characteristic polynomial, will lead to the closed loop system matrix as

State space form

$$A_{CL} = \begin{bmatrix} 0 & 1 & 0 & \cdots & 0 \\ 0 & 0 & 1 & \cdots & 0 \\ \vdots & \ddots & & & \\ \vdots & & \ddots & & 1 \\ 0 & 0 & & \cdots & \\ -\alpha_n & -\alpha_{n-1} & -\alpha_{n-2} & \cdots & -\alpha_1 \end{bmatrix} \cdots \cdots (2)$$

## Pole Placement Control Design: Method - 2

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**Comparing Equation (1) and (2), we arrive at:**

$$\begin{bmatrix} a_n + k_1 = \alpha_n \\ a_{n-1} + k_2 = \alpha_{n-1} \\ \vdots \\ a_1 + k_n = \alpha_1 \end{bmatrix} \Rightarrow \begin{bmatrix} k_1 = (\alpha_n - a_n) \\ k_2 = (\alpha_{n-1} - a_{n-1}) \\ \vdots \\ k_n = (\alpha_1 - a_1) \end{bmatrix}$$

$$K = (\alpha - a) \quad (\text{Row vector form})$$

What if the system is not given in the first companion form?

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Define a transformation  $X = T\hat{X}$

$$\dot{\hat{X}} = T^{-1}\dot{X}$$

$$\dot{\hat{X}} = T^{-1}(AX + Bu)$$

$$\dot{\hat{X}} = (T^{-1}AT)\hat{X} + (T^{-1}B)u$$

Design a  $T$  such that  $T^{-1}AT$  will be in first companion form.

Select  $T = MW$

where  $M \triangleq \begin{bmatrix} B & AB & \dots & A^{n-1}B \end{bmatrix}$  is the controllability matrix

## Pole Placement Control Design: Method - 2

$$W = \begin{bmatrix} a_{n-1} & a_{n-2} & \cdots & a_1 & 1 \\ a_{n-2} & & \ddots & \ddots & 0 \\ & \ddots & \ddots & \cdots & \vdots \\ a_1 & 1 & \cdots & \cdots & \vdots \\ 1 & 0 & \cdots & \cdots & 0 \end{bmatrix}$$

Next, design a controller for the transformed system (using the technique for systems in first companion form).

$$u = -\hat{K}\hat{X} = -(\hat{K}T^{-1})X = -KX$$

*Note: Because of its role in control design as well as the use of  $M$  (Controllability Matrix) in the process, the 'first companion form' is also known as 'Controllable Canonical form'.*

## Pole Placement Design Steps: Method 2: Bass-Gura Approach

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- Check the controllability condition
- Form the characteristic polynomial for  $A$   
 $|sI - A| = s^n + a_1s^{n-1} + a_2s^{n-2} + \dots + a_{n-1}s + a_n$   
find  $a_i$ 's
- Find the Transformation matrix  $T$
- Write the desired characteristic polynomial  
 $(s - \mu_1) \cdots (s - \mu_n) = s^n + \alpha_1s^{n-1} + \alpha_2s^{n-2} + \dots + \alpha_n$   
and determine the  $\alpha_i$ 's
- The required state feedback gain matrix is  
$$K = [(\alpha_n - a_n) \quad (\alpha_{n-1} - a_{n-1}) \quad \cdots \quad (\alpha_1 - a_1)] T^{-1}$$

## Pole Placement Design Steps: Method 3 (Ackermann's formula)

Define  $\tilde{A} = A - BK$

desired characteristic equation is

$$|sI - (A - BK)| = (s - \mu_1) \cdots (s - \mu_n)$$

$$|sI - \tilde{A}| = s^n + \alpha_1 s^{n-1} + \alpha_2 s^{n-2} + \cdots + \alpha_{n-1} s + \alpha_n = 0$$

Caley-Hamilton theorem states that every matrix  $A$  satisfies its own characteristic equation

$$\phi(\tilde{A}) = \tilde{A}^n + \alpha_1 \tilde{A}^{n-1} + \alpha_2 \tilde{A}^{n-2} + \cdots + \alpha_{n-1} \tilde{A} + \alpha_n = 0$$

For the case  $n = 3$  consider the following identities.

$$\mathbf{I} = \mathbf{I}$$

$$\tilde{\mathbf{A}} = \mathbf{A} - \mathbf{BK}$$

$$\tilde{\mathbf{A}}^2 = (\mathbf{A} - \mathbf{BK})^2 = \mathbf{A}^2 - \mathbf{ABK} - \mathbf{BK}\tilde{\mathbf{A}}$$

$$\tilde{\mathbf{A}}^3 = (\mathbf{A} - \mathbf{BK})^3 = \mathbf{A}^3 - \mathbf{A}^2\mathbf{BK} - \mathbf{ABK}\tilde{\mathbf{A}} - \mathbf{BK}\tilde{\mathbf{A}}^2$$

## Pole Placement Design Steps: Method 3: (Ackermann's formula)

Multiplying the identities in order by  $\alpha_3, \alpha_2, \alpha_1$  respectively and adding we get

$$\begin{aligned}
 & \alpha_3 \mathbf{I} + \alpha_2 \tilde{\mathbf{A}} + \alpha_1 \tilde{\mathbf{A}}^2 + \tilde{\mathbf{A}}^3 \\
 &= \alpha_3 \mathbf{I} + \alpha_2 (\mathbf{A} - \mathbf{BK}) + \alpha_1 (\mathbf{A}^2 - \mathbf{ABK} - \mathbf{BK}\tilde{\mathbf{A}}) + \mathbf{A}^3 - \mathbf{A}^2 \mathbf{BK} \\
 & \quad - \mathbf{ABK}\tilde{\mathbf{A}} - \mathbf{BK}\tilde{\mathbf{A}}^2 \\
 &= \alpha_3 \mathbf{I} + \alpha_2 \mathbf{A} + \alpha_1 \mathbf{A}^2 + \mathbf{A}^3 - \alpha_2 \mathbf{BK} - \alpha_1 \mathbf{ABK} - \alpha_1 \mathbf{BK}\tilde{\mathbf{A}} - \mathbf{A}^2 \mathbf{BK} \\
 & \quad - \mathbf{ABK}\tilde{\mathbf{A}} - \mathbf{BK}\tilde{\mathbf{A}}^2 \quad \dots\dots\dots(1)
 \end{aligned}$$

From Caley-Hamilton Theorem for  $\tilde{\mathbf{A}}$

$$\alpha_3 \mathbf{I} + \alpha_2 \tilde{\mathbf{A}} + \alpha_1 \tilde{\mathbf{A}}^2 + \tilde{\mathbf{A}}^3 = \phi(\tilde{\mathbf{A}}) = \mathbf{0}$$

And also we have for  $\mathbf{A}$

$$\alpha_3 \mathbf{I} + \alpha_2 \mathbf{A} + \alpha_1 \mathbf{A}^2 + \mathbf{A}^3 = \phi(\mathbf{A}) \neq \mathbf{0}$$

## Pole Placement Design Steps: Method 3 (Ackermann's formula)

Substituting  $\varphi(\tilde{\mathbf{A}})$  and  $\varphi(\mathbf{A})$  in equation (1) we get

$$\cancel{\varphi(\tilde{\mathbf{A}})} = \varphi(\mathbf{A}) - \alpha_2 \mathbf{B}\mathbf{K} - \alpha_1 \mathbf{B}\mathbf{K}\tilde{\mathbf{A}} - \mathbf{B}\mathbf{K}\tilde{\mathbf{A}}^2 - \alpha_1 \mathbf{A}\mathbf{B}\mathbf{K} - \mathbf{A}\mathbf{B}\mathbf{K}\tilde{\mathbf{A}} - \mathbf{A}^2 \mathbf{B}\mathbf{K}$$

$$0 \rightarrow \varphi(\mathbf{A}) = \mathbf{B}(\alpha_2 \mathbf{K} + \alpha_1 \mathbf{K}\tilde{\mathbf{A}} + \mathbf{K}\tilde{\mathbf{A}}^2) + \mathbf{A}\mathbf{B}(\alpha_1 \mathbf{K} + \mathbf{K}\tilde{\mathbf{A}}) + \mathbf{A}^2 \mathbf{B}\mathbf{K}$$

$$= [\mathbf{B} \quad \mathbf{A}\mathbf{B} \quad \mathbf{A}^2 \mathbf{B}] \begin{bmatrix} \alpha_2 \mathbf{K} + \alpha_1 \mathbf{K}\tilde{\mathbf{A}} + \mathbf{K}\tilde{\mathbf{A}}^2 \\ \alpha_1 \mathbf{K} + \mathbf{K}\tilde{\mathbf{A}} \\ \mathbf{K} \end{bmatrix}$$

Since system is completely controllable inverse of the controllability matrix exists we obtain

$$[\mathbf{B} \quad \mathbf{A}\mathbf{B} \quad \mathbf{A}^2 \mathbf{B}]^{-1} \varphi(\mathbf{A}) = \begin{bmatrix} \alpha_2 \mathbf{K} + \alpha_1 \mathbf{K}\tilde{\mathbf{A}} + \mathbf{K}\tilde{\mathbf{A}}^2 \\ \alpha_1 \mathbf{K} + \mathbf{K}\tilde{\mathbf{A}} \\ \mathbf{K} \end{bmatrix} \dots\dots\dots(2)$$



## Pole Placement Design Steps: Method 3 (Ackermann's formula)

Pre multiplying both sides of the equation (2) with  $[0 \ 0 \ 1]$

$$[0 \ 0 \ 1][\mathbf{B} \mid \mathbf{A}\mathbf{B} \mid \mathbf{A}^2\mathbf{B}]^{-1}\phi(\mathbf{A}) = [0 \ 0 \ 1] \begin{bmatrix} \alpha_2\mathbf{K} + \alpha_1\mathbf{K}\tilde{\mathbf{A}} + \mathbf{K}\tilde{\mathbf{A}}^2 \\ \alpha_1\mathbf{K} + \mathbf{K}\tilde{\mathbf{A}} \\ \mathbf{K} \end{bmatrix} = \mathbf{K}$$

- For an arbitrary positive integer  $n$  ( number of states) *Ackermann's formula* for the state feedback gain matrix  $K$  is given by

$$K = [0 \ 0 \ 0 \ \dots \ \dots \ 1] \begin{bmatrix} B & AB & A^2B & \dots & \dots & \dots & A^{n-1}B \end{bmatrix}^{-1} \phi(A)$$

where  $\phi(A) = A^n + \alpha_1 A^{n-1} + \dots + \alpha_{n-1} A + \alpha_n I$

$\alpha_i$  's are the coefficients of the desired characteristic polynomial

## Choice of closed loop poles : Guidelines

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- Do not choose the closed loop poles far away from the open loop poles, otherwise it will demand high control effort
- Do not choose the closed loop poles very negative, otherwise the system will be fast reacting (i.e. it will have a small time constant)
  - In frequency domain it leads to large bandwidth, and hence noise gets amplified

# Choice of closed loop poles : Guidelines

- Use “Butterworth polynomials”

$$\left(\frac{s}{\omega_o}\right) = (-1)^{\frac{n+1}{2n}} = \left(\underbrace{e^{j(2k+1)\pi}}_{-1}\right)^{\frac{n+1}{2n}} \quad k = 0, 1, 2 \dots$$

$\omega_o$  = a constant (like " natural frequency")

$n$  = system order (number of closed loop poles)

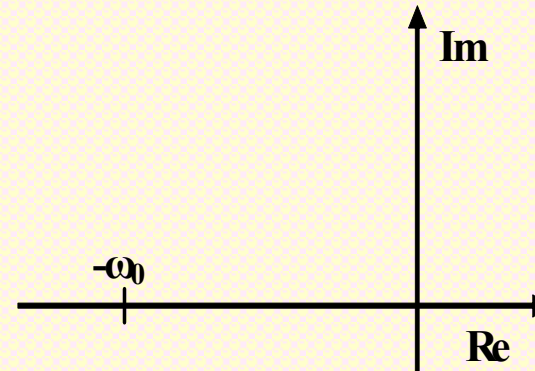
choose only stable poles.

**Example: 1**

let  $n = 1$  only one pole

use  $k = 1$

$$s = \omega_o (\cos \pi + j \sin \pi) = -\omega_o$$



# Choice of closed loop poles : Guidelines

## Example 2:

Let  $n = 2$  we know  $(\cos \theta + j \sin \theta)^m = (\cos m\theta + j \sin m\theta)$

$$\frac{n+1}{2n} = \frac{3}{4}$$

$$s = \omega_0 [\cos((2k+1)3\pi/4) + j \sin((2k+1)(3\pi/4))]$$

$$k = 0 \Rightarrow s_1 = \omega_0 [\cos(3\pi/4) + j \sin(3\pi/4)]$$

stabilizing: accept.

$$k = 1 \Rightarrow s_2 = \omega_0 [\cos(9\pi/4) + j \sin(9\pi/4)]$$

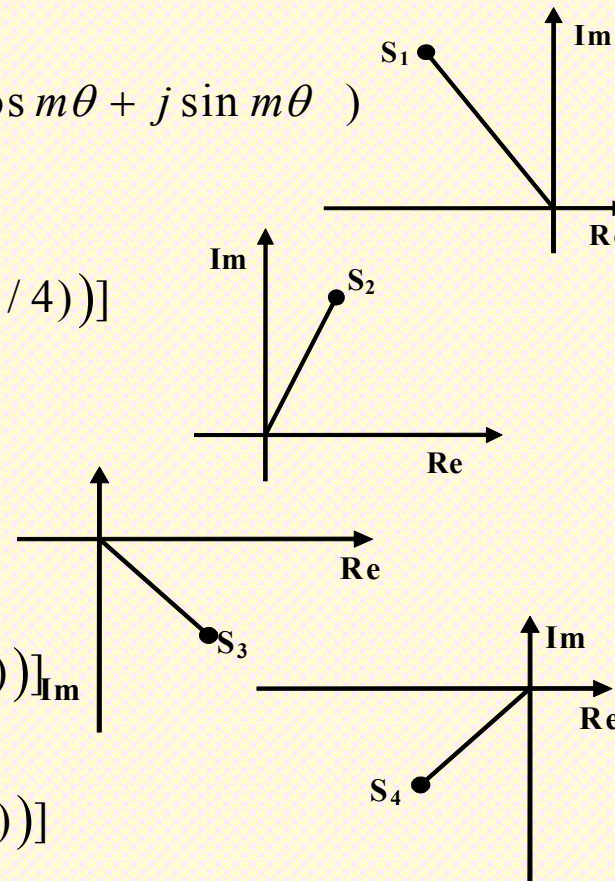
Destabilizing: reject.

$$k = 2 \Rightarrow s_3 = \omega_0 [\cos(15\pi/4) + j \sin(15\pi/4)]$$

Destabilizing: reject.

$$k = 3 \Rightarrow s_4 = \omega_0 [\cos(21\pi/4) + j \sin(21\pi/4)]$$

stabilizing: accept.



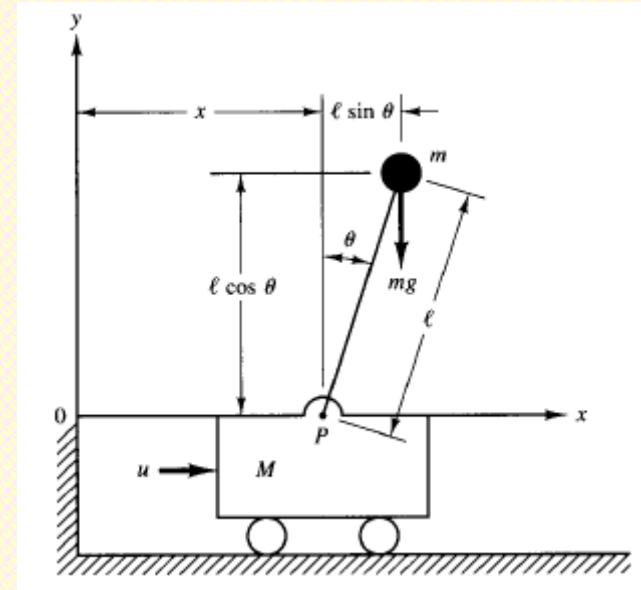


# Example: Inverted Pendulum

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ \frac{M+m}{Ml}g & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 \\ -\frac{m}{M}g & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + \begin{bmatrix} 0 \\ -\frac{1}{Ml} \\ 0 \\ \frac{1}{M} \end{bmatrix} u$$

$$\begin{bmatrix} y_1 \\ y_2 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

$$A = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 20.601 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 \\ -0.4905 & 0 & 0 & 0 \end{bmatrix}; B = \begin{bmatrix} 0 \\ -1 \\ 0 \\ 0.5 \end{bmatrix}; C = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \end{bmatrix}$$



## Step 1: Check controllability

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$$\mathbf{M} = [\mathbf{B} \quad \mathbf{AB} \quad \mathbf{A}^2\mathbf{B} \quad \mathbf{A}^3\mathbf{B}] = \begin{bmatrix} 0 & -1 & 0 & -20.601 \\ -1 & 0 & -20.601 & 0 \\ 0 & 0.5 & 0 & 0.4905 \\ 0.5 & 0 & 0.4905 & 0 \end{bmatrix}$$

$$|\mathbf{M}| \neq 0$$

Hence, the system is controllable.

Step 2: Form the characteristic equation and get  $a_i$ 's

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$$\begin{aligned} |s\mathbf{I} - \mathbf{A}| &= \begin{vmatrix} s & -1 & 0 & 0 \\ -20.601 & s & 0 & 0 \\ 0 & 0 & s & -1 \\ 0.4905 & 0 & 0 & s \end{vmatrix} \\ &= s^4 - 20.601s^2 \\ &= s^4 + a_1s^3 + a_2s^2 + a_3s + a_4 = 0 \end{aligned}$$

$$a_1 = 0, \quad a_2 = -20.601, \quad a_3 = 0, \quad a_4 = 0$$



Step 3: Find Transformation  $\mathbf{T} = \mathbf{M}\mathbf{W}$  and its inverse

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$$\mathbf{W} = \begin{bmatrix} a_3 & a_2 & a_1 & 1 \\ a_2 & a_1 & 1 & 0 \\ a_1 & 1 & 0 & 0 \\ 1 & 0 & 0 & 0 \end{bmatrix} = \begin{bmatrix} 0 & -20.601 & 0 & 1 \\ -20.601 & 0 & 1 & 0 \\ 0 & 1 & 0 & 0 \\ 1 & 0 & 0 & 0 \end{bmatrix}$$

$$\mathbf{T} = \mathbf{M}\mathbf{W} = \begin{bmatrix} 0 & 0 & -1 & 0 \\ 0 & 0 & 0 & -1 \\ -9.81 & 0 & 0.5 & 0 \\ 0 & -9.81 & 0 & 0.5 \end{bmatrix}$$

$$\mathbf{T}^{-1} = \begin{bmatrix} -\frac{0.5}{9.81} & 0 & -\frac{1}{9.81} & 0 \\ 0 & -\frac{0.5}{9.81} & 0 & -\frac{1}{9.81} \\ -1 & 0 & 0 & 0 \\ 0 & -1 & 0 & 0 \end{bmatrix}$$

Step 4: Find  $\alpha_i$ 's from desired poles  $\mu_1, \mu_2, \mu_3, \mu_4$

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$$\mu_1 = -2 + j2\sqrt{3}, \quad \mu_2 = -2 - j2\sqrt{3}, \quad \mu_3 = -10, \quad \mu_4 = -10$$

$$\begin{aligned}(s - \mu_1)(s - \mu_2)(s - \mu_3)(s - \mu_4) &= (s + 2 - j2\sqrt{3})(s + 2 + j2\sqrt{3})(s + 10)(s + 10) \\ &= (s^2 + 4s + 16)(s^2 + 20s + 100) \\ &= s^4 + 24s^3 + 196s^2 + 720s + 1600 \\ &= s^4 + \alpha_1s^3 + \alpha_2s^2 + \alpha_3s + \alpha_4 = 0\end{aligned}$$

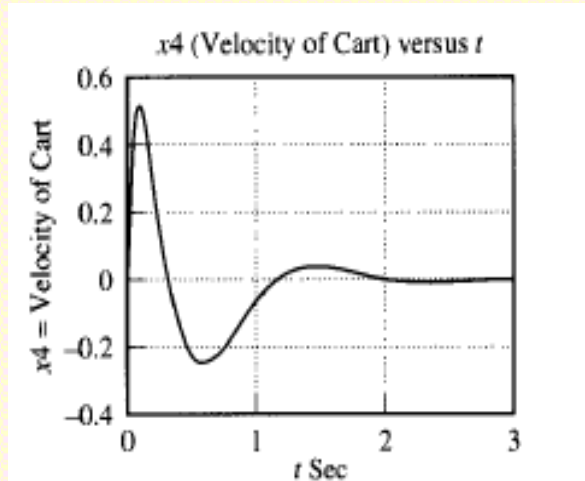
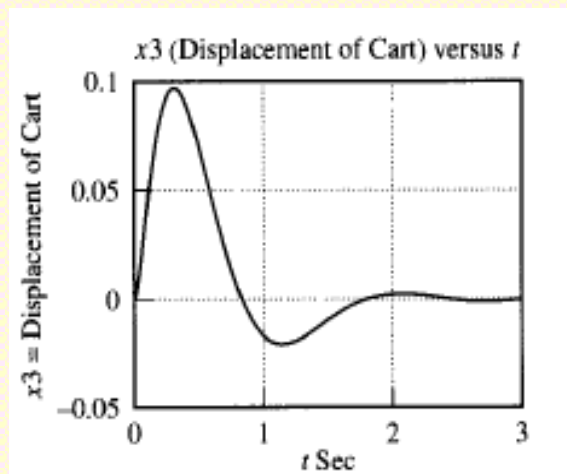
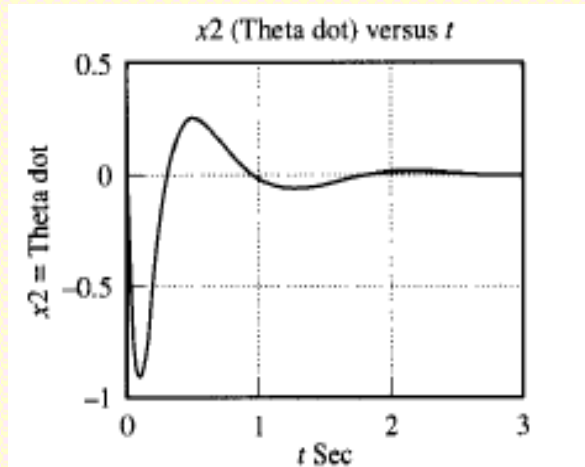
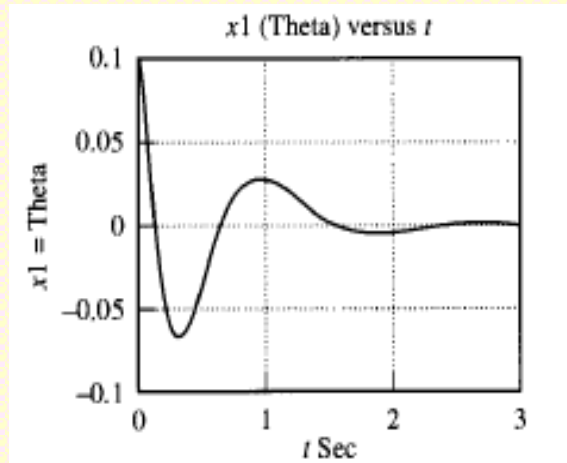
$$\alpha_1 = 24, \quad \alpha_2 = 196, \quad \alpha_3 = 720, \quad \alpha_4 = 1600$$

Step 5: Find State Feed back matrix  $\mathbf{K}$  and input  $u$

$$\begin{aligned}\mathbf{K} &= [\alpha_4 - a_4 \quad \alpha_3 - a_3 \quad \alpha_2 - a_2 \quad \alpha_1 - a_1] \mathbf{T}^{-1} \\ &= [1600 - 0 \quad 720 - 0 \quad 196 + 20.601 \quad 24 - 0] \mathbf{T}^{-1} \\ &= [1600 \quad 720 \quad 216.601 \quad 24] \begin{bmatrix} -\frac{0.5}{9.81} & 0 & -\frac{1}{9.81} & 0 \\ 0 & -\frac{0.5}{9.81} & 0 & -\frac{1}{9.81} \\ -1 & 0 & 0 & 0 \\ 0 & -1 & 0 & 0 \end{bmatrix} \\ &= [-298.1504 \quad -60.6972 \quad -163.0989 \quad -73.3945]\end{aligned}$$

$$u = -\mathbf{K}\mathbf{x} = 298.1504x_1 + 60.6972x_2 + 163.0989x_3 + 73.3945x_4$$

# Time Simulation: Inverted Pendulum



## Choice of closed loop poles : Guidelines

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- Do not choose the closed loop poles far away from the open loop poles. Otherwise, it will demand high control effort.
- Do not choose the closed loop poles very negative. Otherwise, the system will be fast reacting (i.e. it will have a small time constant)  
In frequency domain it leads to a large bandwidth, which in turn leads to amplification of noise!

## Multiple input systems

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- The gain matrix is not unique even for fixed closed loop poles.
- Involved (but tractable) mathematics

$$\dot{X} = AX + BX$$

$$U = -KX$$

$$K = \begin{bmatrix} k_{11} & k_{12} & \cdots & k_{1n} \\ \vdots & \vdots & \cdots & \vdots \\ \vdots & \vdots & \cdots & \vdots \\ k_{m1} & k_{m2} & \cdots & k_{mn} \end{bmatrix}$$

# Multiple input systems: Some tricks and ideas

- Eliminate the need for measuring some  $x_j$  by appropriately choosing the closed loop poles.

$$\textit{Example: } u = \mu_1 x_1 + (\mu_2 - \beta) x_2$$

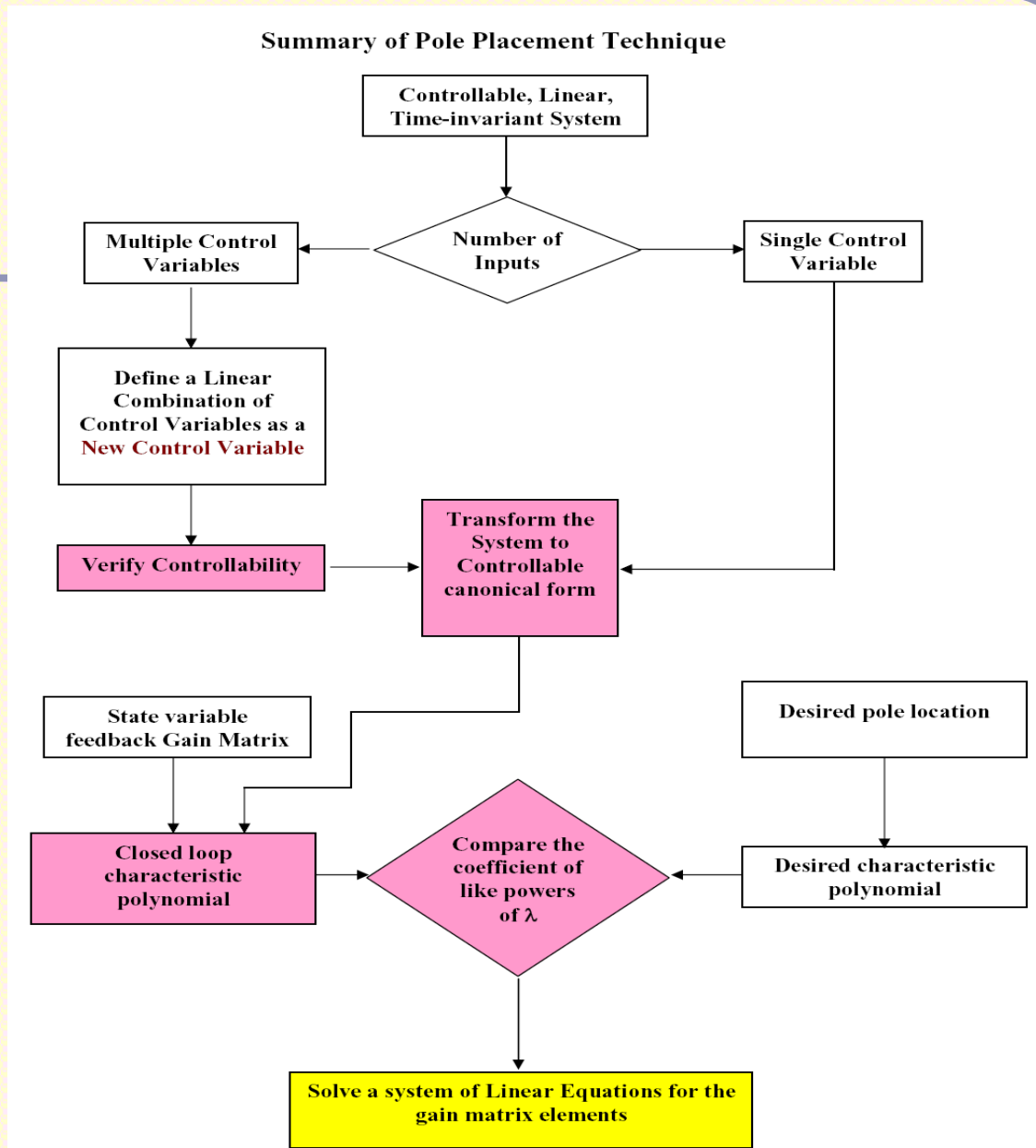
select  $\mu_2 = \beta$  provided  $\beta < 0$

- Relate the gains to proper physical quantities

$$\begin{bmatrix} u_x \\ u_y \end{bmatrix} = \begin{bmatrix} g_{11} & 0 & g_{13} & 0 \\ 0 & g_{22} & 0 & g_{24} \end{bmatrix} \begin{bmatrix} x \\ y \\ \dot{x} \\ \dot{y} \end{bmatrix}$$

- Shape eigenvectors: “Eigen structure assignment control”
- Introduce the idea of optimality: “optimal control”

## Control Allocation:





## References

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- K. Ogata: *Modern Control Engineering*, 3<sup>rd</sup> Ed., Prentice Hall, 1999.
- B. Friedland: *Control System Design*, McGraw Hill, 1986.

**Thanks for the Attention...!**

