#### **Prof. Millind D. Atrey**

Department of Mechanical Engineering, **IIT Bombay** 

Lecture No - 35

### **Earlier Lecture**

- In the earlier lecture, we have seen that radiation is dominant mode of heat transfer in vacuum.
- Evacuated powders are superior in performance than vacuum alone, in **300 K – 77 K**, as radiation heat transfer is comparatively less.
- In an opacified powder, radiation heat transfer is minimized by addition of reflective flakes.
- A tutorial problem is solved to compare the different types of insulations, so far discussed.

# **Outline of the Lecture**

#### **Topic : Cryogenic Insulation (contd)**

- Multilayer Insulation
- Tutorial
- Conclusion

# **Introduction**

In the earlier lecture, we have solved the following tutorial.

**77 K**  $1.2m$  1.6m **300 K**

0.25m

• A spherical **LN2** vessel (**e**=0.8) is as shown. The inner and outer radii are 1.2m and 1.6m respectively. Compare and comment on the heat in leak for the following cases.

• Perlite, Less Vacuum (1.5mPa), Vacuum alone, Vacuum  $+$  10 shields, Evacuated Fine Perlite, 50/50 Cu – Santocel.





- It is clear that opacified powder is the best insulation.
- A heat in leak of 4.41 W to **LN2** would vaporize 2.36 Lit/day as shown in the next slide.

### **Introduction**

- Latent heat of  $LN2 = 200$  kJ/Kg, Density of  $LN2 =$ 807 kg/m3.
- $1m^3 = 1000$  Lit and a day has 24 hours.
- 1 Lit/hr boil off of LN2 is equivalent to



- Hence, 4.41 W of heat vaporizes 0.098 Lit/hr.
- Therefore, the total boil off in 1 day is 2.36 Lit.

<sup>6</sup> **Prof. M D Atrey, Department of Mechanical Engineering, IIT Bombay**

### **Introduction**

- Different types of insulations discussed in the earlier lectures are applicable only to **300 K** to **77 K** temperature range.
- Given that, latent heat of LHe  $= 20.2$ kJ/Kg and density of LHe =  $124.8 \text{ kg/m}^3$ , 1 Lit/hr boil off of LHe is equivalent to 0.7 W.
- The same amount of heat in leak, that is 4.41 W, would vaporize 151.1 Lit of LHe in one day.
- <sup>7</sup> **Prof. M D Atrey, Department of Mechanical Engineering, IIT Bombay** • Therefore, there is a need to develop better insulations for **77 K** to **4 K** temperature range.

# **Types of Insulation**

- Expanded Foam Mass
- Gas Filled Powders & Fibrous Materials Mass
- Vacuum alone Vacuum
- Evacuated Powders Mass + Vacuum
- Opacified Powders Mass + Vacuum + Reflective
- Multilayer Insulation Vacuum + Reflective

<sup>8</sup> **Prof. M D Atrey, Department of Mechanical Engineering, IIT Bombay**

# **Multilayer Insulation**

• Multilayer Insulation (MLI) was first developed by Petersen of Sweden in the year 1951.



- It consists of alternate layers of
	- High reflecting shields or foils

Shield Spacer

- Separated by low conductivity spacers
- And a very good vacuum.

# **Multilayer Insulation**

The high reflecting shields are generally made of either Al, Cu or Aluminized Mylar.



• Aluminum sheet of 6µm thickness is commonly used at low temperatures.



• In order to improve mechanical strength and ease of application, plastic materials like Mylar and Kapton are coated with aluminum.

# **Multilayer Insulation**

• Low conductivity spacers are made of coarse silk or nylon net.



• Very often, substances like glass fiber, silica fiber, low density foam or fiber glass mat are also used.

Shield Spacer

- Most common materials among fibers are Dexiglas and Tissuglas.
- One layer of MLI is defined as one sheet of reflective shield + one sheet of spacer material.

<sup>11</sup> **Prof. M D Atrey, Department of Mechanical Engineering, IIT Bombay**

# **Multilayer Insulation**

• Each component of this insulation is designed for a particular function.



- Radiation shields Foils with high reflectivity reduce radiant heat transfer.
- Spacers Nylon spacers with very low thermal conductivity reduce conduction.
- Vacuum Residual gas conduction, convection are minimized using vacuum.

### **Types of MLI**

• MLIs are classified according to the type of spacers used.



- $\rightarrow$  Multiple Resistance Spacers: Fibers are arranged in a parallel fashion to minimize contact area.
- → Point Contact Spacers: A grid of nylon spheres is used to separate adjacent radiation shields.

# **Types of MLI**

• Continuing further, we have



- → Single Component MLI: Reflective shields are crinkled or embossed to minimize contact area. These MLI do not use any spacer material.
- $\rightarrow$  Composite Spacers: Few spacers consist of two or more materials. Each material has a specific function to perform.

- Typically, thickness of each layer is 6 μm.
- Residual gas conduction inside an insulation depends on residual pressure of the gas.
- For an optimum performance, the usual levels of vacuum, that is maintained around an MLI, are in the range of **7.5x10-5** torr.

# **Applications**

• For an optimum performance, MLI is placed perpendicular to direction of heat flow.



- The insulation performance is a function of following parameters.
	- Applied compressive load
	- Number of shields
	- Gas type and its pressure
	- Size and number of perforations
	- Operating temperature

### **Applications**







Courtesy: TIFR, Mumbai



- The adjacent figure shows variation of  $k_A$  with residual gas pressure for a typical Multilayer Insulation.
- Insulation layer density is 24 layers/cm with boundary temperatures maintained at 300 K and 90 K.



- It is clear that,  $k_A$  is independent of residual gas pressure between atmospheric and 15 torr.
- With lowering of pressure, **k**<sup>a</sup> is directly proportional to residual gas pressure.
	- The variation is almost linear on a logarithmic chart as shown in the figure.



- The mode of heat transfer in this regime is mainly due to free molecular conduction or residual gas conduction.
- With further lowering of pressures, that is less than 10<sup>-3</sup> torr,  $k_A$  remains fairly constant.

# **Multilayer Insulation**

- MLI bulk density ( $\rho_a$ ) is an important parameter of the insulation. It depends on
	- Thickness of each reflective shield t<sub>r</sub>
	- Density of each reflective shield **ρ<sup>r</sup>**
	- Mass per unit area of the spacer  $-S<sub>s</sub>$
	- Layer density per unit thickness **N/Δx**
- The total mass per unit area is given by  $(S_s + \rho_r t_r)$
- Density being mass per unit volume, for **N** layers, **ρ<sup>a</sup>** is given by  $\rho_a = \left( S_s + \rho_r t_r \right) \frac{N}{\Lambda}$  $\rho_a = (S_s + \rho_s)$

*x*

∆

# **Multilayer Insulation**

• The apparent thermal conductivity (μW/mK) and layer density (layer/cm) of few commonly used MLI are as shown. Residual gas pressure is 10<sup>-5</sup> torr with end temperatures as **77 K** and **300 K**.



# **Apparent Thermal Cond.**

• For an evacuated MLI, heat is transferred by radiation and solid conduction. *c*

 $T_{h}$   $\Lambda$   $T$ 

• For 1 layer, net heat transferred (O<sub>net</sub>) is

$$
Q_{net} = Q_{Rad} + Q_{Solid\,Cond}
$$

$$
Q_{net} = F_e F_{1\rightarrow 2} \sigma A \left( T_h^4 - T_c^4 \right) + \frac{k_c A \left( T_h - T_c \right)}{\Delta x}
$$

- $F_a$  Effective emissivity of the Shields
- $F_{1\rightarrow 2}$  Shape factor
- **A, ∆x**  Contact area and Width
- **k<sub>c</sub>** Effective thermal conductivity

<sup>23</sup> **Prof. M D Atrey, Department of Mechanical Engineering, IIT Bombay**

 $T_{h}$   $\Lambda$   $T$ 

*c*

# **Apparent Thermal Cond.**

$$
Q_{net} = F_e F_{1\rightarrow 2} \sigma A \left( T_h^4 - T_c^4 \right) + \frac{k_c A \left( T_h - T_c \right)}{\Delta x}
$$



Combining above equations, we have

$$
Q_{net} = \sigma \left(\frac{e}{2-e}\right) A \left(T_h^4 - T_c^4\right) + h_c A \left(T_h - T_c\right)
$$

- **e** Emissivity of the shield
- $\cdot$  **h<sub>c</sub>** Thermal conductance per unit area

# **Apparent Thermal Cond.**

$$
Q_{net} = \sigma \left(\frac{e}{2-e}\right) A \left(T_h^4 - T_c^4\right) + h_c A \left(T_h - T_c\right)
$$



\n- Let 
$$
k_A
$$
 be apparent thermal conductivity of insulation. Therefore,  $Q_{\text{net}}$  is
\n

$$
Q_{net} = \frac{k_A A (T_h - T_c)}{\Delta x}
$$

• Equating above two equations and rearranging, we have

$$
\frac{k_A A(T_h - T_c)}{\Delta x} = A(T_h - T_c) \bigg( \sigma \bigg( T_h^2 + T_c^2 \bigg) \bigg( T_h + T_c \bigg) \bigg( \frac{e}{2 - e} \bigg) + h_c \bigg)
$$

# **Apparent Thermal Cond.**

The apparent thermal conductivity  $(k_A)$  is



*c*

$$
k_A = \Delta x \left( \sigma \left( T_h^2 + T_c^2 \right) \left( T_h + T_c \right) \left( \frac{e}{2 - e} \right) + h_c \right)
$$

For **N** layers, we have

$$
k_A = \frac{\Delta x}{N} \left( \sigma \left( T_h^2 + T_c^2 \right) \left( T_h + T_c \right) \left( \frac{e}{2 - e} \right) + h_c \right)
$$

- where,
	- $T_h$ ,  $T_c$  Boundary temperatures
	- **N/Δx** Layer density

# **Apparent Thermal Cond.**





- With an initial increase in layer density, the decrease in radiation heat transfer is more than the increase in solid conduction.
- Hence,  $k_A$  of the insulation decreases.

# **Apparent Thermal Cond.**



- With further increase in layer density,  $k_A$  increases due to an increase in solid conduction (**h**<sub>c</sub>).
- Therefore,  $k_A$  goes through a minima and then rises as shown in the figure.

**294 K**

**77 K**

**4 K** 2.0 m

 $0.6 \, \text{m}$  1.6m

2.4 $m$ 

MLI

# **Tutorial**

• Consider a spherical **LHe** vessel shielded with **LN2** bath. The radii of the spherical shells are as shown in the figure. MLI (24 layers/cm) is applied at each stage. Calculate the boil off/day of LN2 and LHe.

Given that emissivity of shield is 0.05. Solid conductance of spacer is 0.0851 W/m2K (assumed constant). Also, neglect neck conduction.

# **Tutorial**

#### **Given**

Multi Layer Insulation Operating LN2 boil off : 294 K to 77 K Temperature LHe boil off : 77 K to 4 K Emissivity of Shield : 0.05 Number of layers : 24/cm Solid conductance : 0.0851 W/m2K

#### **Calculate**

Boil off of LN2 and LHe on per day basis.

### **Tutorial**

Calculation of  $k_A$  for LN2 (294 K to 77 K)

•  $\Delta x/N = (1/2400)$ , h<sub>c</sub>=0.0851, e=0.05, T<sub>h</sub>, T<sub>c</sub>.

$$
k_A = \left(\frac{\Delta x}{N}\right) \left(h_c + \sigma e \left(T_h^2 + T_c^2\right) \left(\frac{T_h + T_c}{2 - e}\right)\right)
$$

$$
k_A = \left(\frac{1}{2400}\right) \left(0.0851 + \frac{(5.669)(10^{-8})(0.05)(92365)(371)}{(2 - 0.05)}\right)
$$

$$
k_{A} = 56.2 \,\mu W / mK
$$

**77 K**

**4 K** 2.0 m

 $0.6 \; \mathrm{h}$ 

2.4 $m$ 

MLI

### **Tutorial**

Heat in leak for LN2 (294 K to 77 K) •  $T_{h}$ ,  $T_{c}$ ,  $k_{A} = 56.2 \mu W/mK$ ,  $R_{1} = 2.4 m$ , 294 K R<sub>2</sub>=2.0m, ∆T=(294-77)=217.





 $Q = 1.84W$ 

<sup>32</sup> **Prof. M D Atrey, Department of Mechanical Engineering, IIT Bombay**

### **Tutorial**

Boil off of LN2 (294 K to 77 K)

- Latent heat of LN2 = 200 kJ/Kg, Density of LN2 = 807 kg/m3.
- 1 Lit/hr of LN2 is equivalent to 44.83 W.
- Hence, 1.84 W of heat vaporizes 0.041 Lit/hr.
- Therefore, the total boil off of LN2 in 1 day is 0.985 Lit.

### **Tutorial**

Calculation of  $k_A$  for LHe (77 K to 4 K)

•  $\Delta x/N = (1/2400)$ , h<sub>c</sub>=0.0851, e=0.05, T<sub>h</sub>, T<sub>c</sub>.

$$
k_A = \left(\frac{\Delta x}{N}\right) \left(h_c + \sigma e \left(T_h^2 + T_c^2\right) \left(\frac{T_h + T_c}{2 - e}\right)\right)
$$

$$
k_A = \left(\frac{1}{2400}\right) \left(0.0851 + \frac{(5.669)(10^{-8})(0.05)(5945)(81)}{(2 - 0.05)}\right)
$$

$$
k_{A} = 35.7 \,\mu W / mK
$$

<sup>34</sup> **Prof. M D Atrey, Department of Mechanical Engineering, IIT Bombay**

**77 K**

**4 K** 2.0 m

 $0.6 \; \mathrm{h}$ 

2.4 $m$ 

MLI

### **Tutorial**

Heat in leak for LHe (77 K to 4 K)

•  $T_{h}$ ,  $T_{c}$ ,  $k_{A} = 35.7 \mu W/mK$ ,  $R_{1} = 1.6 m$ , 294 **K**  $R_2 = 0.6$ m,  $\Delta T = (77-4) = 73$ .





 $Q = 0.031W$ 

### **Tutorial**

Boil off for LHe (77 K to 4 K)

- Latent heat of LHe  $= 20.2$  kJ/Kg, Density of LHe  $=$ 124.8 kg/m<sup>3</sup>.
- 1 Lit/hr of LHe is equivalent to 0.7W.
- Hence, 0.031 W of heat vaporizes 0.044 Lit/hr.
- Therefore, the total boil off of LHe in 1 day is 1.062 Lit.

### **Tutorial**



# **Conclusion**

- Cryogenic vessels need insulation to minimize all modes of heat transfer.
- **k**A is calculated based on all the possible modes of heat transfer.
- In an expanded foam, heat is transferred only by solid conduction. With decrease in mean cell diameter,  $k_A$  decreases. With an increase in bulk density,  $k_A$  increases.
- <sup>38</sup> **Prof. M D Atrey, Department of Mechanical Engineering, IIT Bombay** In a gas filled powder or a fibrous insulation, heat is transferred by gas and solid conductions.

# **Conclusion**

- In vacuum, radiation is the dominant mode of heat transfer. It is minimized by using radiation shields.
- In an evacuated powder, heat is transferred by free molecular conduction, solid conduction and radiation. At low pressures and temperatures, solid conduction dominates radiation.
- In an opacified powder, the radiation heat transfer is minimized by addition of reflective flakes.
- <sup>39</sup> **Prof. M D Atrey, Department of Mechanical Engineering, IIT Bombay** • MLI consist of alternate layers of high reflecting shields and low conducting spacers.

# **Conclusion**

- Multi Layer Insulations are more effective in **77 K** to **4 K** temperatures, when provided with a good vacuum.
- There is an optimum layer density, at which  $k_A$  of the insulation is minimum.

### **Thank You!**

<sup>41</sup> **Prof. M D Atrey, Department of Mechanical Engineering, IIT Bombay**