

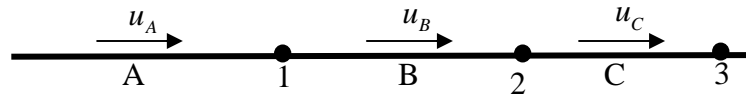
Fluid flow

Q1.

In the one-dimensional constant-density situation shown below, the momentum equations for u_B and u_C can be written as follows:

$$u_B = 5 + 2.5(p_1 - p_2)$$

$$u_C = 5 + 7.5(p_2 - p_3)$$



The boundary conditions are as: $u_A = 15$ (all values are given in consistent units). Obtain the values for p_1 and p_2 , following the SIMPLE calculation procedure. Comment on the uniqueness of your solution.

Solution

Assume $p_1^* = p_2^* = p_3^* = 0$

Solving momentum equation, we get

$$u_B^* = 5$$

Continuity equation becomes

$$\frac{du}{dx} = 0$$

Integrating within a control volume as shown

$$u_e = u_w$$

Continuity equation for grid point 1

$$u_A = u_B$$

$$u_A = 5 + 2.5(p_1' - p_2')$$

$$15 = 5 + 2.5(p_1' - p_2')$$

(16)

Continuity equation for grid point 2

$$u_B = u_C$$

$$5 + 2.5(p_1' - p_2') = u_C$$

$$5 + 2.5(p_1' - p_2') = 15$$

(17)

No unique solutions of (16) and (17)

Solutions are guided by initial guess for iteration, depicting the relative nature of p as a numerically computed variable.

Let $p_1' = 0$

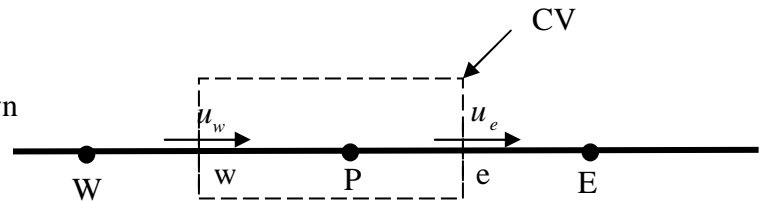
$$p_2' = -4$$

$$p_1 = p_1^* + p_1' = 0$$

$$p_2 = p_2^* + p_2' = -4$$

$$u_B = u_B^* + 2.5(p_1' - p_2') = 5 + 2.5(0 + 4) = 15$$

Solution is converged.

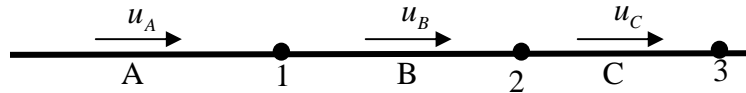


Q2.

In the one-dimensional constant-density situation below, the momentum equations for u_B and u_C can be written as follows:

$$u_B = 5 + 2.5(p_1 - p_2)$$

$$u_C = 5 + 7.5(p_2 - p_3)$$



The boundary conditions are as: $u_B = 15$, $p_3 = 10$ (all values are given in consistent units).

- (i) Write the continuity equations for the regions AB and BC and hence derive the corresponding pressure correction equations.
- (ii) Starting with guesses for p_1 and p_2 , follow the SIMPLE procedure to obtain converged values of p_2 , u_B and u_C .

Solution

Assume $p_1^* = p_2^* = 0$

Solving momentum equations, we get

$$u_B^* = 5$$

$$u_C^* = 5 + 7.5(0 - 10) = -70$$

$$u_e = u_e^* + d_e(p'_P - p'_E)$$

$$u_w = u_w^* + d_w(p'_W - p'_P)$$

Continuity equation becomes

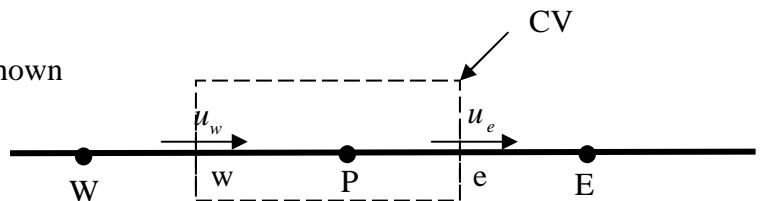
$$\frac{du}{dx} = 0$$

Integrating within a control volume as shown

$$u_e = u_w$$

$$u_e^* + d_e(p'_P - p'_E) = u_w^* + d_w(p'_W - p'_P)$$

$$(d_e + d_w)p'_P = d_e p'_E + d_w p'_W + (u_w^* - u_e^*)$$

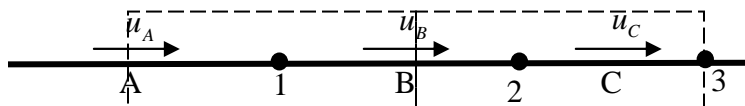


For point 1 as shown

$$(d_B + d_A)p'_1 = d_B p'_2 + d_A p'_1 + (u_A^* - u_B^*)$$

$$2.5p'_1 = 2.5p'_2 + 10$$

or, $p'_1 = p'_2 + 4$ CV 1 CV 2 (18)



For point 2 as shown

$$(d_C + d_B) p'_2 = d_C p'_3 + d_B p'_1 + (u_B^* - u_C^*)$$

$$10 p'_2 = 2.5 p'_1 + (5 + 70)$$

or, $4 p'_2 = p'_1 + 30$ (19)

Solving Eqs (18) and (19), we get

$$p'_1 = \frac{46}{3}$$

$$p'_2 = \frac{34}{3}$$

$$u_B = u_B^* + d_B (p'_1 - p'_2) = 5 + 2.5 \left(\frac{46}{3} - \frac{34}{3} \right) = 15$$

$$u_C = u_C^* + d_C (p'_2 - p'_3) = -70 + 7.5 \left(\frac{34}{3} - 0 \right) = 15$$

$$p_1 = p_1^* + p'_1 = \frac{46}{3}$$

$$p_2 = p_2^* + p'_2 = \frac{34}{3}$$

$$b = u_w^* - u_e^*$$

For control volume 1

$$b = u_A^* - u_B^* = 0$$

For control volume 2

$$b = u_B^* - u_C^* = 0$$

This is the continuity satisfying velocity field.

Q3.

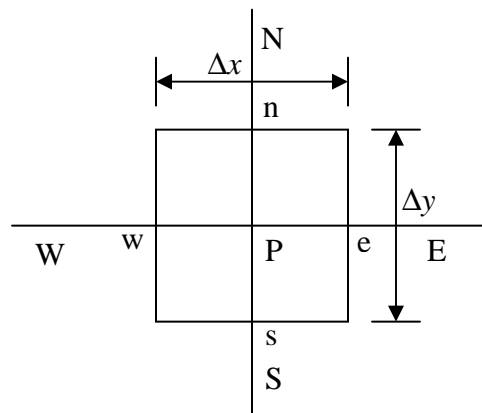
In the two dimensional situation shown, the following quantities are given:

$u_w = 50$, $v_s = 20$, $p_N = 0$, $p_E = 10$. The flow is steady and the density is uniform. The momentum equations for u_e and v_n are given by:

$$u_e = d_e (p_P - p_E)$$

$$v_n = d_n (p_P - p_N)$$

where the constants d_e and d_n are given by $d_e = 1$, $d_n = 0.6$. The control volume shown has $\Delta x = \Delta y$. Use SIMPLE algorithm to obtain the values of u_e , v_n and p_P .



Solution

Let $p_W^* = 20$, $p_S^* = 20$, $p_P^* = 15$

Solving momentum equations, we get

$$u_e^* = d_e (p_P^* - p_E^*) = (1)(15 - 10) = 5$$

$$v_n^* = d_n (p_P^* - p_N^*) = 0.6(15 - 0) = 9$$

Continuity equation becomes

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

Integrating within a control volume as shown

$$\int_s^n \int_w^e \frac{\partial u}{\partial x} dx dy + \int_w^e \int_s^n \frac{\partial v}{\partial y} dy dx = 0$$

$$(u_e - u_w) \Delta y + (v_n - v_s) \Delta x = 0$$

$$u_e^* + d_e (p_P' - p_E') + u_w^* + d_w (p_W' - p_P') + v_n^* + d_n (p_P' - p_N') + v_s^* + d_s (p_S' - p_P') = 0$$

$$(d_e + d_w + d_n + d_s) p_P' = d_e p_E' + d_w p_W' + d_n p_N' + d_s p_S' + (u_w^* - u_e^*) + (v_s^* - v_n^*)$$

$$1.6 p_P' = d_e p_E'^0 + d_w p_W'^0 + d_n p_N'^0 + d_s p_S'^0 + (50 - 5) + (20 - 9)$$

$$p_P' = \frac{56}{1.6} = 35$$

$$u_e = u_e^* + d_e (p_P' - p_E') = 5 + 1(35 - 0) = 40$$

$$v_n = v_n^* + d_n (p_P' - p_N') = 9 + 0.6(35 - 0) = 30$$

$$p_P = p_P^* + p_P' = 15 + 35 = 50$$

$$b = u_w^* - u_e^* + v_s^* - v_n^* = 50 - 40 + 20 - 30 = 0$$

Solution is converged.

Q.4

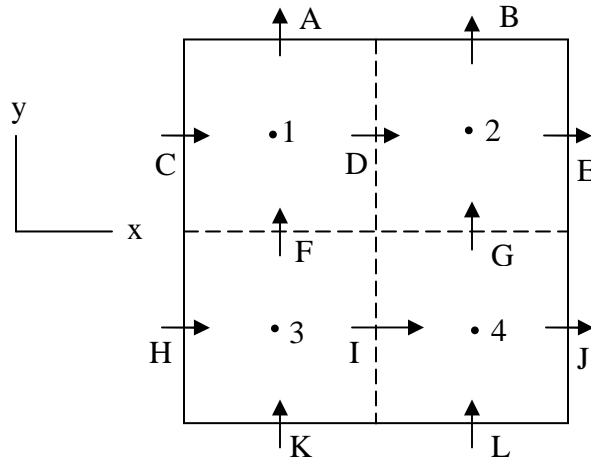
A steady, uniform-density, 2-D flow is to be calculated on the square grid shown below. The boundary velocities are given as; $v_A = 30$, $v_B = 40$, $u_C = 100$, $u_E = 50$, $u_H = 200$, $u_J = 210$, $v_K = 0$ and $v_L = 20$. Among these numbers, there is some doubt about correctness of the value of u_J . If all other numbers are correct, what should be the correct value of u_J ?

The internal velocities are governed by simplified momentum equations given by:

$$u_D = 70 + 0.5(p_1 - p_2) \quad u_I = 10 + 0.7(p_3 - p_4)$$

$$v_F = 30 + 0.5(p_3 - p_1) \quad v_G = 18 + 0.8(p_4 - p_2)$$

Write discretized continuity equation for each control volume. Derive the discretization equation for pressure by substituting from momentum equations, following SIMPLER calculation procedure. Solve the pressure equations to obtain p_1, p_2, p_3 and p_4 . Hence obtain values of u_D, u_I, v_F and v_G .



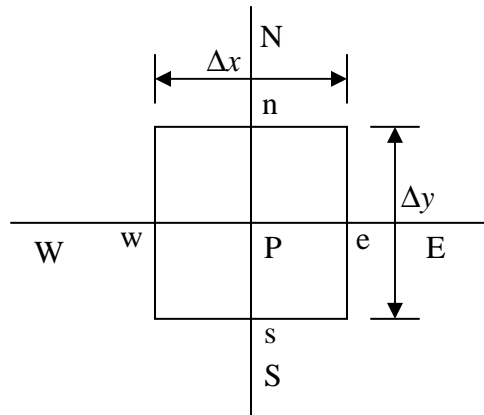
Solution

From the conservation of mass, we have

$$\sum \text{rate of inflow} = \sum \text{rate of outflow}$$

$$100 + 200 + 0 + 20 = 30 + 40 + 50 + u_j$$

or, $u_j = 200$



Continuity equation becomes

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

Integrating within a control volume as shown, it becomes

$$\int_s^n \int_w^e \frac{\partial u}{\partial x} dx dy + \int_w^e \int_s^n \frac{\partial v}{\partial y} dy dx = 0$$

$$(u_e - u_w) \Delta y + (v_n - v_s) \Delta x = 0$$

Momentum equations can be expressed as

$$u_e = \hat{u}_e + d_e (p_P - p_E)$$

$$u_w = \hat{u}_w + d_w (p_P - p_W)$$

$$v_n = \hat{v}_n + d_n (p_P - p_N)$$

$$v_s = \hat{v}_s + d_s (p_P - p_S)$$

$$(d_e + d_w + d_n + d_s) p_P = d_e p_E + d_w p_W + d_s p_S + d_n p_N + \hat{u}_w - \hat{u}_e + \hat{v}_s - \hat{v}_n$$

For grid point 1

$$d_e = d_D = 0.5, d_w = d_C = 0$$

$$d_s = d_F = 0.5, d_n = d_A = 0$$

$$p_1 = 0.5 p_2 + 0.5 p_3 + (100 - 70) + (30 - 30)$$

$$p_1 = 0.5 p_2 + 0.5 p_3 + 30$$

For grid point 2

$$d_e = d_E = 0, d_w = d_D = 0.5$$

$$d_s = d_G = 0.8, d_n = d_B = 0$$

$$1.3 p_2 = 0.5 p_1 + 0.8 p_4 + (70 - 70) + (18 - 40)$$

$$1.3 p_2 = 0.5 p_1 + 0.8 p_4 - 2$$

For grid point 3

$$d_e = d_E = 0, d_w = d_D = 0.5$$

$$d_s = d_G = 0.8, d_n = d_B = 0$$

$$1.3 p_2 = 0.5 p_1 + 0.8 p_4 + (70 - 70) + (18 - 40)$$

$$1.2 p_3 = 0.7 p_4 + 0.5 p_1 + 160$$

For grid point 4

$$d_e = d_E = 0, d_w = d_D = 0.5$$

$$d_s = d_G = 0.8, d_n = d_B = 0$$

$$1.3 p_2 = 0.5 p_1 + 0.8 p_4 + (70 - 70) + (18 - 40)$$

$$1.5 p_4 = 0.7 p_3 + 0.8 p_2 - 188$$

Choose $p_4 = 0$ (reference)

$$p_1 = 160$$

$$p_2 = 60$$

$$p_3 = 200$$

$$u_D^* = 70 + 0.5(p_1^* - p_2^*) = 70 + 0.5(160 - 60) = 120$$

$$u_I^* = 10 + 0.7(p_3^* - p_4^*) = 10 + 0.7(200 - 0) = 150$$

$$v_F^* = 30 + 0.5(p_3^* - p_1^*) = 30 + 0.5(200 - 160) = 50$$

$$v_G^* = 18 + 0.8(p_4^* - p_2^*) = 18 + 0.8(0 - 60) = -30$$

Pressure correction equation becomes

$$(d_e + d_w + d_n + d_s) p'_P = d_e p'_E + d_w p'_W + d_s p'_S + d_n p'_N + \underbrace{(u_w^* - u_e^*) + (v_s^* - v_n^*)}_b$$

Top left CV:

$$b = (100 - 120) + (50 - 30) = 0$$

Top right CV:

$$b = (120 - 50) + (-30 - 40) = 0$$

Bottom left CV:

$$b = (200 - 150) + (0 - 50) = 0$$

Bottom right CV:

$$b = (150 - 200) + (20 + 30) = 0$$

This is the converged solution.