

# Vibrations of Structures

## Module III: Vibrations of Beams

### Exercises

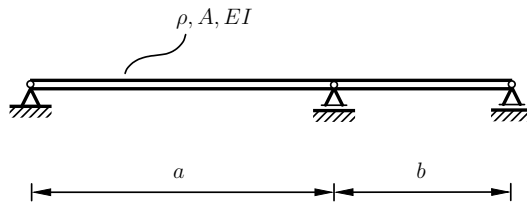


Figure 1: Exercise 1

1. A uniform Euler-Bernoulli beam of length  $a + b$ , flexural stiffness  $EI$  and linear density  $\rho A$  is supported, as shown in Fig. 1. Plot the variation of the first few eigenfrequencies for variation of  $b$  in the range  $(0, a)$ .

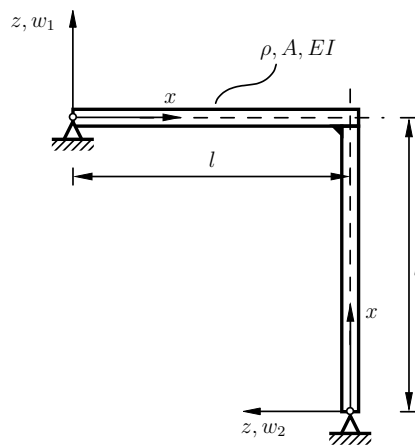


Figure 2: Exercise 2

2. The frame shown in Fig. 2 is made by welding two uniform homogeneous Euler-Bernoulli beams infinitely stiff in tension. Determine the first few eigenfrequencies and the corresponding modes of vibration of the frame.

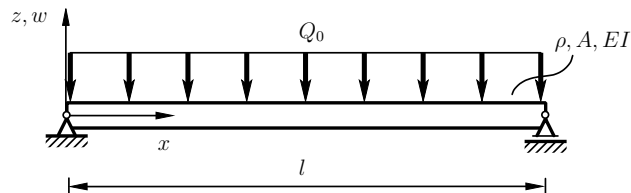


Figure 3: Exercise 3

3. A simply-supported uniform beam is loaded by a constant distributed force  $q(x, t) = Q_0$ , as shown in Fig. 3. Determine the motion of the beam when the force is suddenly removed at  $t = 0$ .

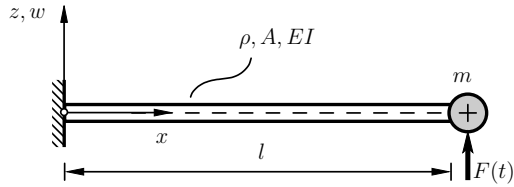


Figure 4: Exercise 4

4. Determine the steady-state response of a uniform cantilever beam with end-mass when excited by a force  $F(t) = F_0 \cos \Omega t$ , as shown in Fig. 4.

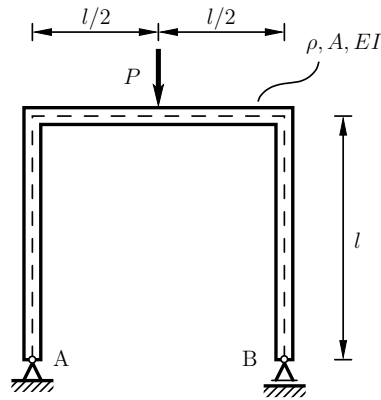


Figure 5: Exercise 5

5. A frame shown in Fig. 5 is made of three uniform beams each having flexural rigidity  $EI$ , linear density  $\rho A$ . A constant force  $P$  acts at the center of the horizontal beam as shown. Estimate the first eigenfrequency of the frame  $\omega_1(P)$  as a function of the force  $P$ . Re-calculate the first eigenfrequency if the frame is pinned at the support B.
6. A uniform beam, pinned at one end, is released from rest from a horizontal position, as shown in Fig. 6. The free-end A of the beam hits a rigid edge when the beam reaches a vertical position. Determine the reaction forces at the ends A and B as functions time.
7. A simply-supported beam of rectangular cross-section has a constant width  $b$ , while the height varies from  $h/2$  to  $h$  over its length  $l$ . The material has density  $\rho$ , and Young's modulus  $E$ . Estimate the first two modal frequencies and modes of vibration.
8. A uniform Euler-Bernoulli beam is fixed to a small hub rotating at a constant angular speed  $\Omega$  in the vertical plane, as shown in Fig. 7, has a Lagrangian

$$\mathcal{L} = \frac{1}{2} \int_0^l [\rho A [(\Omega x + w_{,t})^2 + \Omega^2 w^2] - T w_{,x}^2 - EI w_{,xx}^2] dx,$$

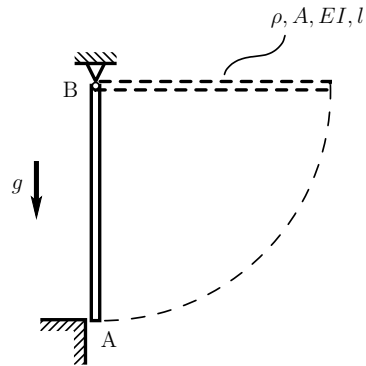


Figure 6: Exercise 6

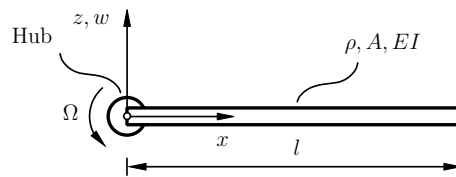


Figure 7: Exercise 8

where  $dT/dx = \rho A \Omega^2 x$ . Derive the equation of motion for transverse vibrations of the beam. Using the Ritz method, estimate the first few eigenfrequencies of the beam, and plot their variation with  $\Omega$ .